
Contents

Certificate	iii
Declaration	v
Copyright	vii
Acknowledgement	ix
Table of Contents	xi
List of Figures	xv
List of Tables	xxi
Nomenclature	xxiii
Abstract	xxvii
1 Introduction	1
1.1 Assessment of energy resources	2
1.2 Conventional solar air heater: An outlook	3
1.3 Literature review on design modification of SAHs	5
1.4 New efficient designs of SAH	8
1.5 Objective of thesis	12
1.6 Structure of thesis	12
2 Efficient Design of Curved Solar Air Heater Integrated with Semi-down Turbulators	13
2.1 Introduction	13
2.2 Numerical domain	15
2.2.1 Geometry specification and parametric details	15
2.2.2 Mesh description and grid independence of numerical model	17
2.2.3 Boundary conditions	19
2.2.4 Governing equations and physical terms	20

2.2.5	Assessment of magnitude of solar radiation received on curved SAH	22
2.3	Experimental validation of numerical model of curved SAH	22
2.4	Results and discussions	23
2.5	Exergy analysis	28
2.6	Development of Correlations	35
2.7	Conclusions	37
3	Efficient Designs of Double-pass Curved Solar Air Heaters	39
3.1	Introduction	39
3.2	Numerical model	43
3.2.1	<i>Computational flow domain</i>	43
3.2.2	<i>Mesh generation, grid-independent, and time-independent study</i>	43
3.2.2.1	<i>Grid independent test</i>	45
3.2.2.2	<i>Time independent study</i>	45
3.2.3	<i>Boundary condition</i>	46
3.2.4	<i>Governing equation</i>	47
3.2.5	<i>Experimental validation of the model</i>	48
3.3	Results and discussions	49
3.3.1	<i>Circular vs. semicircular ribs: which is better?</i>	49
3.3.2	<i>The effect of rib diameter</i>	51
3.3.3	<i>Effect of absorber plate location</i>	57
3.4	Correlation development	59
3.4.1	<i>Correlation for Nu</i>	60
3.4.2	<i>Correlation for f</i>	61
3.5	Conclusions	64
4	Performance Characteristics of a New Curved Double-pass Counter Flow Solar Air Heater	65
4.1	Introduction	65
4.2	Computational fluid dynamics model	68
4.2.1	Description of numerical domain and operating parameters	68
4.2.2	Boundary condition	70
4.2.3	Governing equation	71
4.2.4	Performance parameters	72
4.2.5	Mesh description, grid independence and time independence	74
4.2.5.1	Mesh independence study	74
4.2.5.2	Time independent study	74
4.2.6	Validation with experimental results	75
4.3	Results and discussion	77
4.3.1	Performance comparison: counter vs. parallel curved DPSAH	77

4.3.1.1	Thermal performance of smooth curved SAHs	77
4.3.1.2	Thermal performance of roughened curved SAHs	78
4.3.1.3	Hydraulic performance of curved SAHs	82
4.4	Development of correlation for Nu and f	87
4.4.1	Correlation for Nu	87
4.4.2	Correlation for f	89
4.5	Conclusions	91
5	Investigations for Efficient Design of a New Counter Flow Double-pass Curved Solar Air Heater	93
5.1	Introduction	93
5.2	Problem description and numerical methodology	95
5.2.1	<i>Geometry of computational domain</i>	96
5.2.2	<i>Governing equation and data reduction</i>	96
5.2.3	<i>Grid generation and boundary condition</i>	100
5.2.3.1	Grid generation	100
5.2.3.2	Boundary and initial conditions	100
5.3	Validation of numerical model	101
5.4	Results and discussion	102
5.4.1	<i>Flow visualization</i>	103
5.4.2	Optimization of arched baffle pitch	105
5.4.3	<i>Optimization of arched baffle angle</i>	107
5.5	Regression correlation for roughened curve CDPSAH with baffle	110
5.5.1	<i>Regression correlation for Nusselt number</i>	111
5.5.2	<i>Regression correlation for friction factor</i>	114
5.6	Conclusions	116
6	Effect of Channel Designs and Its Optimization for Enhanced Thermo-hydraulic Performance of Solar Air Heater	117
6.1	Introduction	118
6.2	Description of problem	122
6.3	The numerical model	123
6.3.1	<i>Grid generation and independence test</i>	123
6.3.2	<i>Boundary conditions</i>	124
6.3.3	<i>Governing equations and turbulence model</i>	124
6.3.4	<i>Performance parameters</i>	125
6.3.5	<i>Validation of computational model</i>	127
6.4	Results and Discussion	128
6.4.1	<i>Results of different cross-sectional SAHs</i>	128

6.4.2	<i>Results of the best cross-section shape with corrugated SAH</i>	133
6.4.3	<i>Empirical correlation development for Nu and f</i>	142
6.5	Conclusions	143
7	Conclusions and Recommendations	147
7.1	Conclusions	147
7.2	Major contributions of thesis	151
7.3	Assumptions and limitations of the present investigation	152
7.4	Recommendation for future work	153
	Bibliography	155
	Appendix A	169
	Appendix B	171
	Author's Personal Profile and Publication List	173

List of Figures

1.1	Projections of global primary energy consumption for 2020 to 2050.	2
1.2	Trends in renewable energy from 2000 to 2020.	3
1.3	Schematic diagram of conventional solar air heater.	4
1.4	Physical representation of the performance improvement methods in the solar air heater.	6
1.5	Schematic diagram of curved SAH integrated with semi-down turbulator (i) Half-triangular, (ii) Half-trapezoidal and (iii) Quarter-circle.	10
1.6	Schematic diagram of parallel curved DPSAH	10
1.7	Schematic diagram of counter curved DPSAH.	11
1.8	Schematic diagram of counter curved DPSAH with strategic placement of arched deflectors in lower channel.	11
1.9	Schematic diagram of 3D view semi-ellipse cross-section with wavy absorber plate	11
2.1	Previous investigations based on solar air heater straight flow channel equipped with various shape of ribs in two different arrangements of ribbed absorber plate: (a) down ribs and (b) up/bottom ribs.	16
2.2	Geometry of a curved SAH having 25° curvature angle with the half-triangular (i.e. (i)) grooved absorber plate. The three different shape of ribs have been considered in the study are shown: (i) half-triangular, (ii) half-trapezoidal and (iii) quarter-circle, and analyzed individually to investigate thermo-hydraulic performance.	17
2.3	Mesh of the computational flow domain of curved SAH equipped with half-triangular shape ribs.	18
2.4	Outlet air temperature variation (T_o) with solar radiation intensity (I) at the mass flow rate of 0.0172 kg/s.m ²	23

2.5	Variation of temperature factor, $\frac{(T_o-T_i)}{T}$, of curved SAH equipped with different shape of ribs for Re in the range 11000-15000, at $q = 1000 \text{ W/m}^2$	25
2.6	Shows Nusselt number variation for different shapes of ribs for Re in the range 11000 – 15000, at $q = 1000 \text{ W/m}^2$	26
2.7	Flow velocity profiles of (a) curved smooth SAH and (b) flat smooth SAH at different axial locations along the duct height for Re in the range 11000 at $q = 1000 \text{ W/m}^2$	27
2.8	Friction factor variation for various shape of ribs for Re in the range 11000 – 15000, at $q = 1000 \text{ W/m}^2$	28
2.9	Thermohydraulic performance $\frac{(T_o-T_i)}{T} / f$, variation of different shape of ribs for Re in the range 11000 – 15000, at $q = 1000 \text{ W/m}^2$	29
2.10	Plot $\left(\frac{Nu}{\Delta P}\right)$ vs. Re at $q = 1000 \text{ W/m}^2$ for various shape of ribs.	30
2.11	Variation of effectiveness of different shape of ribs with Re , at $q = 1000 \text{ W/m}^2$	31
2.12	Demonstrates variation of exergy recovery of curved and flat- SAH devices with respect to temperature factor of different shape of ribs for the range of Re 11000-15000, at $q = 1000 \text{ W/m}^2$	32
2.13	Demonstrates second law efficiency (η_{II}) variation of curved and flat- SAH devices with respect to temperature factor of different shape of ribs for the range of Re 11000 – 15000, at $q = 1000 \text{ W/m}^2$	33
2.14	(a) Vorticity, (b) turbulent dissipation rate and (c) temperature contours of the curved SAH equipped with quarter-circle ribs having $e_r/H = 0.125$ and $e_r/b_r=1$, half-trapezoidal ribs having $e_{tp}/H=0.125$ and half-triangular ribs having $e_t/H=0.125$ and $e_t/b_t=1$, respectively, for the Reynolds number of 11,000, at $q = 1000 \text{ W/m}^2$; (d) and (e) showing the Nusselt number and temperature contours at various regions for the best performing half-trapezoidal ribs at the middle section of curved and flat SAH, respectively.	34
2.15	Plot of Nusselt number along the absorber length of best performing curved half-trapezoidal ribbed SAH having $e_{tp}/H= 0.125$ and smooth curved SAH for the Reynolds number 11000, at $q = 1000 \text{ W/m}^2$. Magnified views of velocity contours are also shown at different axial locations along the duct height. Notice how Nu continuously decrease in both the cases, however, Nu shoots up at the location of ribs.	36
2.16	Variation of $\ln A_o$ vs. $\ln \frac{e_r}{H}$	37
2.17	Comparison of Nusselt number values obtained numerically and derived correlation.	38
3.1	Schematic diagram of curved DPSAH with variable relative location of the absorber plates (y/H). Pitch P is kept constant in all the designs.	44
3.2	Two-dimensional unstructured mesh of SAH-II with semicircular metallic ribs.	45

3.3	(a) Comparison of Nusselt number at various Reynolds number; contour of velocity magnitude of (b) SAH-II with semicircular ribs and (c) SAH-II with circular ribs at $Re = 10,000$ and $d/H = 0.25$	50
3.4	A plot of friction factor (f) versus Reynold number (Re) for a constant value of relative roughness height (d/H) of 0.15	51
3.5	(a) Variation of Nusselt number with relative roughness height, (b) Variation of local heat transfer coefficient with position in SAH-I	51
3.6	Variation of local wall shear stress in SAH-II with semicircular and circular ribs at high $Re = 10,000$ and $d/H = 0.25$	52
3.7	Comparison of experimental results with numerical values of outlet air temperature w. r.t solar radiation.	53
3.8	Variation of Nusselt number with Reynolds number for SAH-II.	54
3.9	Contours of turbulent kinetic energy at $Re = 5000$ for fixed value of P/H and different value of relative roughness height (a) $d/H = 0.1$, (b) $d/H = 0.15$, (c) $d/H = 0.20$ and (d) $d/H = 0.25$	55
3.10	Variation of the effectiveness with Reynolds number (a) SAH-I, (b) SAH-II, and (c) SAH-III.	56
3.11	The effect of Reynold number on friction factor for different SAH (a) SAH-II, and (b) SAH-III.	57
3.12	Temperature contour at $Re = 10,000$ at fixed value of $d/H = 0.25$, $P/H = 0.75$ and different relative location of absorber plate (a) SAH-I (i.e., $y/H = 0.25$) (b) SAH-II (i.e., $y/H = 0.50$), and (c) SAH - III (i.e., $y/H = 0.75$).	58
3.13	The variation of mass flow rate with outlet temperature (a) Lower channel (b) Upper channel.	59
3.14	Variation of temperature with position (along with the duct height) at the central location of different SAH with semicircular rib at a fixed value of $Re = 10,000$ and $d/H = 0.25$. The location along the duct height is shown in the inset	60
3.15	Variation of $\ln(Nu)$ as a function of $\ln(Dn)$	62
3.16	Variation of $\ln(C_0)$ with $\ln(D_n)$	62
3.17	Comparison of numerical and predicted values of Nusselt number.	63
3.18	Comparison of numerical and predicted values of the friction factor.	63
4.1	A schematic of 2D numerical domain of (a) roughened counter curved DPSAH and (b) roughened parallel curved DPSAH.	67
4.2	Mesh of a curve roughened counter DPSAH equipped with asymmetric semicircular turbulators.	75
4.3	Comparison of experimental and CFD results of outlet air temperature w.r.t solar radiation.	78

4.4	A plot of Nu with respect to Re at a constant heat flux.	79
4.5	Air temperature distribution along the duct height at mid-section of smooth counter and smooth parallel curved DPSAH at $Re = 10000$	81
4.6	Effect of d/H on Nu for counter and parallel curved DPSAH for a constant value of Re and P/H	82
4.7	Contours of TKE of roughened counter and parallel curved DPSAH at $Re = 10000$ for a constant value P/H and distinct values of relative roughness height (a) $d/H = 0.1$, (b) $d/H = 0.1$, (c) $d/H = 0.15$, (d) $d/H = 0.15$, (e) $d/H = 0.20$, (f) $d/H = 0.20$, (g) $d/H = 0.25$ and (h) $d/H = 0.25$, flow is from left to right.	83
4.8	A contour plot of temperature for a constant value of Re and d/H of 10000 and 0.25 roughened (a) counter (b) parallel curved DPSAH.	84
4.9	A plot of thermal effectiveness versus relative roughness height.	84
4.10	Contour of velocity magnitude at constant value of Re and d/H of roughened (a) Counter (b) Parallel curved DPSAH.	85
4.11	Effect of relative roughness height on thermal hydraulic efficiency for fixed value of Re and P/H of 10000 and 0.75.	86
4.14	A plot of $\ln(A_0)$ vs $\ln(d/H)$	86
4.12	Variation of wall shear stresses with longitudinal length of absorber plate in roughened SAHs.	87
4.13	A plot of $\ln(Nu)$ vs $\ln(Re)$	88
4.15	A plot of $\ln(f)$ vs $\ln(Re)$	89
4.16	A plot of $\ln(C_0)$ vs $\ln(d/H)$	90
4.17	Comparison of predicted vs numerical values of Nusselt number	90
4.18	Comparison of predicted and numerical values of friction factor	91
5.1	Schematic diagram of longitudinal cross section of a double-pass counter flow SAH. (a) Note the strategic placement of arched deflectors in lower channel that directs the fluid towards absorber plate. In second row, left figure show the geometric parameters of the curved SAH. The right figure shows the geometric parameters of the curved baffles, (b) roughened counter DPSAH (i.e., $P/d = 0$) without baffles. Computational model is developed for both the system	97
5.2	Generated mesh on roughened curve CDPSAH	100
5.3	Boundary condition used in the numerical simulation	102
5.4	Observation of difference between temperature of air at inlet and outlet obtained from CFD and experimental results in a curved SAH. Note that better agreement is observed at higher solar radiation.	103
5.5	Velocity streamline and direction of vortices in curve CDPSAH with baffle ($P/d = 6$ and $\alpha/90 = 0.5$) at fixed Re of 10000.	104

5.6	A plot of Nu with P/d for different range of Reynolds numbers. Increase and decrease in Nusselt number variation signifies there lies a optimum relative pitch where thermal performance is optimum. Note that lines connecting the values are not a curve fit. It is drawn just to show the trend. This is true for all the figures in this section.	106
5.7	The variation of thermal effectiveness (ϵ) with relative baffle pitch (P/d).	107
5.8	variation along the longitudinal length of absorber plate in lower channel of CDP-SAH at $Re = 10000$	108
5.9	Variation of friction factor ratio (f_{wb}/f_{wob}) vs relative baffle pitch (P/d).	109
5.10	Variation of Nu with baffle angle $\alpha/90$	110
5.11	Plot of friction factor ratio (f_{wb}/f_{wob}) with relative baffle angle ($\alpha/90$).	111
5.12	Contour plots of turbulent kinetic energy at $Re = 6000$ and $P/d = 6$ for (a) $\alpha/90 = 1$, (b) $\alpha/90 = 0.83$, (c) $\alpha/90 = 0.67$, (d) $\alpha/90 = 0.5$, (e) $\alpha/90 = 0.33$ and (f) for $P/d = 0$ i.e. without baffles.	112
5.13	Variation of local velocity along the duct height (along aa' line) at a longitudinal distance of 1515 mm at $Re = 6000$ and $P/d = 6$. Zones are marked in the right panel schematic figure.	113
5.14	Variation of $\ln(Nu)$ with $\ln(Re)$	114
5.15	Variation of predicted and numerical value of Nu	115
5.16	Comparison of numerical and forecasted values of friction factor.	115
6.1	Schematic diagram of (a) Cross-sectional view (b) 3D view with wavy absorber plate.	123
6.2	View of mesh generated for the rectangular cross-section SAH.	124
6.3	Comparison among numerical, experimental and predicted results of Nusselt number for smooth duct.	128
6.4	Comparison among numerical, experimental and predicted results of friction factor for smooth duct.	129
6.5	Variation of temperature factor, $\left(\frac{T_0-T_i}{T}\right)$ of various cross-sections of SAH duct for the range of Re $11000 - 19000$	130
6.6	Variation of local heat transfer coefficient along the length of absorber plate of different cross-sectional SAHs for given values of Re	131
6.7	Central turbulent intensity distribution along the length of SAH with different cross-section for given values Reynolds number.	132
6.8	Variation in effectiveness (ϵ) with respect to temperature factor, $\left(\frac{T_0-T_i}{T}\right)$ for various cross-sections of SAH duct for the range of Re $11000 - 19000$	133
6.9	Variation of Nu with respect to Re in the range of $11000 - 19000$, at the fixed value heat flux of 500 W/m^2	134

6.10	Wall y^+ distribution along the length of absorber plate for different cross-sectional solar air heater at given values of Reynolds number	135
6.11	Variation of ΔP for the range of Re 11000-19000 for different cross-sections of SAH duct.	136
6.12	Variation of Nu vs Re for different values of A/D_h and λ/D_h	137
6.13	Variation Nu/Nu_s vs Re for different values of A/D_h and λ/D_h	138
6.14	Variation of thermal effectiveness with temperature factor for different values of A/D_h and fixed value of λ/D_h for a range of Re 11000 – 19000.	139
6.15	Contour plots of temperature at $Re = 19000$ and $A/D_h = 0.12$ for (a) $\lambda/D_h = 1.6$, (b) $\lambda/D_h = 1.2$ and (c) $\lambda/D_h = 0.8$	140
6.16	The contour of TKE at $Re = 11000$ and $\lambda/D_h = 0.8$ for (a) $A/D_h = 0.04$ (b) $A/D_h = 0.08$ and (c) $A/D_h = 0.12$	140
6.17	The contour plots of velocity magnitude at $Re = 11000$ and $\lambda/D_h = 0.8$ for (a) $A/D_h = 0.04$ (b) $A/D_h = 0.08$ and (c) $A/D_h = 0.12$	141
6.18	Plots of f with Re for distinct values of A/D_h and λ/D_h in corrugated SAH with semi-ellipse cross-section.	142
6.19	Plots of f/f_s with Re for distinct values of A/D_h and λ/D_h in corrugated SAH with semi-ellipse cross-section.	143
6.20	Plots of Nu/f with Re for distinct values of A/D_h and λ/D_h in corrugated SAH with semi-ellipse cross-section.	144
6.21	The contour plots of static pressure at $Re = 11000$ and $\lambda/D_h = 0.8$ for (a) $A/D_h = 0.04$ (b) $A/D_h = 0.08$ and (c) $A/D_h = 0.12$	145
6.22	Variation of local wall shear stress along the length of duct of SAH with dissimilar values of A/D_h for fixed value of Re and λ/D_h of 11000 and 0.8 respectively.	146
A.1	Trend variation of $\ln(Nu)$ versus $\ln(Re)$ for SAH having curved design equipped with quarter-circle ribs.	170
A.2	Variation of $\ln(A_o)$ vs. $\ln\left(\frac{e_r}{H}\right)$	170

List of Tables

2.1	Description of design and flow parameters.	18
2.2	Grid independence test.	19
2.3	Air properties at 300 K.	19
3.1	List of the relevant literature of double-pass solar air heater having various roughness geometry and flow arrangement. Notice that rectangular designs of DPSAHs.	40
3.1	List of the relevant literature of double-pass solar air heater having various roughness geometry and flow arrangement. Notice that rectangular designs of DPSAHs.	41
3.1	List of the relevant literature of double-pass solar air heater having various roughness geometry and flow arrangement. Notice that rectangular designs of DPSAHs.	42
3.2	Range of parameters for numerical modeling.	44
3.3	Grid independent study	46
3.4	Time independent study.	46
3.5	Constant thermo-physical properties of working fluid and absorber plate employed for numerical simulation.	46
4.1	Relevant literature on double pass solar air heater have various shapes of absorber plate, extended surface and flow pattern.	69
4.2	Range of geometrical and operating parameters.	70
4.3	Properties of materials for numerical simulation [6, 74].	71
4.4	Grid independent study	75
4.5	Time independent test	76
4.6	Experimental validation of pressure drop across the curve SAHs	76
4.7	Thermal hydraulic efficiency comparison among smooth curved designs SAHs.	79
5.1	The operating and geometrical parameters considered for the numerical analysis:	97

5.2	Properties of materials for numerical simulation [6, 74].	101
6.1	The list of previous studies to augment thermal performance using different cross-sectional shape and corrugated shape of absorber surface in the flow channel. . .	118
6.1	The list of previous studies to augment thermal performance using different cross-sectional shape and corrugated shape of absorber surface in the flow channel. . .	119
6.1	The list of previous studies to augment thermal performance using different cross-sectional shape and corrugated shape of absorber surface in the flow channel. . .	120
6.1	The list of previous studies to augment thermal performance using different cross-sectional shape and corrugated shape of absorber surface in the flow channel. . .	121
6.2	Range of geometrical and flow parameters.	122

Nomenclature

a	Absorption coefficient (m^{-1})
A	Amplitude of wavy surface (m)
A_c	Cross-sectional area of duct (m^2)
A_{eff}	Effective area of absorber plate (m^2)
A/D_h	Relative roughness amplitude
b_r	Width of quarter-circle groove(m)
b_t	Width of half-triangular groove(m)
b_{tp}	Top width of half-trapezoidal groove(m)
b_{tpd}	Bottom width of half-trapezoidal groove(m)
C	Perimeter of duct (m)
C_p	Specific heat at constant pressure (J/kgK)
d	Rib diameter (m)
d/H	Relative roughness height
D_h	Hydraulic diameter (m)
D_n	Dean number
e_r	Quarter-circle groove height(m)
e_r/H	Relative quarter-circle groove height ratio
e_t	Half-triangular groove height(m)
e_t/H	Relative half-triangular groove height ratio
e_{tp}	Half-trapezoidal groove height (m)
e_{tp}/H	Relative half-trapezoidal groove height ratio
f	Friction factor
f/f_s	Friction factor enhancement ratio
g	Acceleration due to gravity (m/s^2)
G_b	Turbulent kinetic energy due to buoyancy (J/kg)
G_k	Turbulent kinetic energy due to mean velocity gradient (J/kg)
h	Convective heat transfer coefficient (W/m^2K)
H	Height of duct (m)
I	Solar irradiance (W/m^2)
k_a	Thermal conductivity of air(W/mK)
k_g	Thermal conductivity of glass(W/mK)
k_i	Thermal conductivity of insulation(W/mK)
L	Length of absorber plate (m)
\dot{m}	Mass flow rate (kg/s)
n	Refractive index of medium
Nu	Nusselt Number

Nu/Nu_s	Nusselt number enhancement ratio
P	Pitch of groove or rib or baffle(m)
P/e_r	Relative pitch ratio of quarter-circle groove
P/e_t	Relative pitch ratio of half-triangular groove
P/e_{tp}	Relative pitch ratio of half- trapezoidal groove
P/d	Relative arched baffle pitch
P/H	Relative roughness of pitch
ΔP	Pressure drop (N/m^2)
Pr	Prandtl number
q	Heat flux (W/m^2)
Q_u	Useful heat rate gain (W)
R_c	Curvature radius (m)
R_i	Inner radius of arched baffle (m)
R_o	Outer radius of arched baffle (m)
Re	Reynolds number
TF	Temperature factor
TI	Turbulent intensity
T_i	Inlet temperature (K)
T_o	Outlet air temperature (K)
T_m	Mean bulk temperature (K)
T_p	Absorber plate temperature (K)
T_{sky}	Sky temperature (K)
U	Mean velocity of fluid (m/s)
V_W	Wind velocity (m/s)
W	Width of SAH (m)
W/H	Aspect ratio of SAH

Greek symbols

$a/90$	Relative baffle angle
α	Absorptivity
β	Curvature angle of SAH
ϵ	Thermal effectiveness
ϵ	Emissivity
λ	The wavelength of the wavy absorber (m)
μ	Dynamic viscosity (Ns/m^2)
η	Efficiency
ρ	Density of air (kg/m^3)
τ	Transmissivity of glass
ω	Solid angle

Subscripts

a	Air
c	Curvature
bm	Bulk mean temperature
∞	Free stream condition
g	glass cover
i	Inlet section
l	Lower channel
u	upper channel
o	Outlet section
p	absorber plate

Abbreviations

SAH	Solar air heater
TKE	Turbulent kinetic energy
SPSAH	Single-pass solar air heater
DPSAH	Double pass solar air heater
CDPSAH	Counter double -pass solar air heater

