

## Appendix B

### Simple Calculations Using EES Software

#### B.1 Calculation of Reynolds number

$\rho$  = density of oil=900[kg/m<sup>3</sup>]

$\mu$  = coefficient of dynamic viscosity of thermal oil =0.027[kg/m-s]

$d$  = helical coil tube diameter=0.005[m]

$Q$  = flow rate of oil through helical coil tube=5.88[l/min]\*convert(l/min,m<sup>3</sup>/s)

$A_c$  = area of cross section of tube of flow=( $\pi/4$ )\* $d^2$

$v$  = velocity of flow= $Q/A$

$Re$  = Reynolds number= $\rho*v*d/\mu$

#### B.2 Calculation of Prandtl number

$C_p$  = specific heat capacity of oil =2061[J/kg-K]

$k_{con}$  = thermal conductivity of oil=0.165[W/m-k]

$b$  = pitch of the helical coil tube=24.38[mm]\*convert(mm,m)

$\gamma$  = dimensionless pitch of coil= $b/(\pi*D_c)$

$Pr$  = Prandtl Number= $\mu*C_p/k_{con}$

### B.3 Calculation of curvature ratio

$$D_c = 0.025[\text{m}]$$

$$\Delta = \text{curvature ratio} = d/D_c$$

### B.4 Calculation of Dean number

$$De = Re * (\Delta)^{0.5}$$

### B.5 Calculation of helix number

$$He = De / (1 + (\gamma)^2)^{0.5}$$

### B.6 Experimental determination of h

$$m_{\dot{}} = 0.0882[\text{kg/s}] \text{ "Mass flow rate of thermal oil "}$$

$$C_p = 2061[\text{J/kg}\cdot\text{K}] \text{ "Specific heat capacity of HTF"}$$

$$k_c = 400[\text{W/m}\cdot\text{K}] \text{ "Thermal conductivity of copper tube"}$$

$$A_r = 0.1137[\text{m}^2] \text{ "Receiver surface area"}$$

$$t = 0.001[\text{m}] \text{ "Thickness of the coil tube"}$$

$$d = 0.005[\text{m}] \text{ " Inner diameter of the coil tube"}$$

$$L = 5.172[\text{m}] \text{ " Total length of the helical coil"}$$

$$T_i = 53.3 + 273[\text{K}] \text{ " Inlet fluid temperature"}$$

$$T_e = 61.5 + 273[\text{K}] \text{ "Exit fluid temperature"}$$

$$T_{so} = 67.5 + 273[\text{K}] \text{ " Outer surface temperature of receiver tube"}$$

$$T_m = (T_e + T_i) / 2 \text{ "Mean fluid temperature"}$$

$$Q_u = m_{\dot{}} * C_p * (T_e - T_i) \text{ "Useful heat gain"}$$

$$Q_u = (2 * \pi * k_c * L * (T_{si} - T_{so})) / (\ln((d + 2 * t) / d))$$

$h=Q_u/(A_r*(T_{si} - T_m))$  "Convective heat transfer coefficient between coil fluid interface"

## B.7 Calculation of thermal efficiency

$m_{dot}=0.0882[\text{kg/s}]$  "Mass flow rate of thermal oil"

$C_p=2061[\text{J/kg-k}]$  "Specific heat capacity of HTF"

$k_c=400[\text{W/m-k}]$  "Thermal conductivity of copper tube"

$A_r=0.1137[\text{m}^2]$  "Receiver surface area"

$t=0.001[\text{m}]$  "Thickness of the coil tube"

$d=0.005[\text{m}]$  " Inner diameter of the coil tube"

$L=5.172[\text{m}]$  " Total length of the helical coil"

$A_a=2.2487[\text{m}^2]$  " Aperture area of PTC"

$T_i=53.3+273[\text{K}]$  " Inlet fluid temperature"

$T_e=61.5+273[\text{k}]$  "Exit fluid temperature"

$T_{so}=67.5+273[\text{K}]$  " Outer surface temperature of receiver tube"

$T_m=(T_e+T_i)/2$  "Mean fluid temperature"

$Q_u= m_{dot}*C_p*(T_e-T_i)$  "Useful heat gain"

$Q_u= (2*\pi*k_c*L*(T_{si}-T_{so}))/(\text{LN}((d+2*t)/d))$

$h=Q_u/(A_r*(T_{si} - T_m))$  "Convective heat transfer coefficient between coil fluid interface"

$I_b=766[\text{W/m}^2]$  "beam solar radiation"

$\text{Eta}=Q_u/(A_a*I_b)$  "Efficiency of experimental setup"

## B.8 Calculation of pressure drop across the length of helical coil

$m_{dot}=0.0882[\text{kg/s}]$  "Mass flow rate of thermal oil"

$C_p=2061[\text{J/kg-K}]$  "Specific heat capacity of HTF"

$k_c=400[\text{W/m-K}]$  "Thermal conductivity of copper tube"

$A_r=0.1137[\text{m}^2]$  "Receiver surface area"

$t=0.001[\text{m}]$  "Thickness of the coil tube"

$d=0.005[\text{m}]$  " Inner diameter of the coil tube"

$D_c = 0.025[\text{m}]$  "Curvature diameter of helical coil tube"

$L=5.172[\text{m}]$  " Total length of the helical coil"

$A_a=2.2487[\text{m}^2]$  " Aperture area of PTC"

$v=4.991[\text{m/s}]$  "Velocity of flow"

$\rho=900[\text{kg/m}^3]$  " Density of fluid"

$Re=831.9$  "Reynolda number"

$f_{\text{coil}} = (344*(D_c/d)^{-0.5})/(1.56+\text{LOG}_{10}(Re*(D_c/d)^{-0.5}))^{5.33}$  "Friction factor of helical coil tube"

$\text{DELTA}_P=f_{\text{coil}}*(L/d)*0.5*\rho*v^2$  " Required pressure drop across the length of helical coil"

## B.9 Simple Solutions

### Unit Settings: [J]/[K]/[bar]/[kg]/[degrees]

$A_c = 0.00001963 \text{ [m}^2\text{]}$	$b = 0.02438$	$C_p = 2061 \text{ [J/kg-K]}$	$d = 0.005 \text{ [m]}$
$He = 355.3$	$k_{con} = 0.165 \text{ [W/m-k]}$	$\mu = 0.027 \text{ [kg/m-s]}$	$Pr = 337.3$
$De = 372$	$\delta = 0.2$	$D_c = 0.025 \text{ [m]}$	$\text{gamma} = 0.3104$
$Q = 0.000098 \text{ [m}^3\text{/s]}$	$Re = 831.9$	$\rho = 900 \text{ [kg/m}^3\text{]}$	$v = 4.991 \text{ [m/s]}$

### Unit Settings: [J]/[K]/[bar]/[kg]/[degrees]

$A_r = 0.1137 \text{ [m}^2\text{]}$	$C_p = 2061 \text{ [J/kg-k]}$	$d = 0.005 \text{ [m]}$	$h = 1293 \text{ [W/m}^2\text{-K]}$
$t = 0.001 \text{ [m]}$	$T_e = 334.5 \text{ [K]}$	$T_i = 326.3 \text{ [K]}$	$T_m = 330.4 \text{ [K]}$
$k_c = 400 \text{ [W/m-k]}$	$L = 5.172 \text{ [m]}$	$\dot{m} = 0.0882 \text{ [kg/s]}$	$Q_u = 1491 \text{ [J]}$
$T_{si} = 340.5 \text{ [K]}$	$T_{so} = 340.5 \text{ [K]}$		

### Unit Settings: [J]/[K]/[bar]/[kg]/[degrees]

$A_a = 2.249 \text{ [m}^2\text{]}$	$A_r = 0.1137 \text{ [m}^2\text{]}$	$C_p = 2061 \text{ [J/kg-k]}$	$d = 0.005 \text{ [m]}$
$L = 5.172 \text{ [m]}$	$\dot{m} = 0.0882 \text{ [kg/s]}$	$Q_u = 1491 \text{ [J]}$	$t = 0.001 \text{ [m]}$
$T_{so} = 340.5 \text{ [K]}$			
$\eta = 0.8654$	$h = 1293 \text{ [W/m}^2\text{-K]}$	$h_b = 766 \text{ [W/m}^2\text{]}$	$k_c = 400 \text{ [W/m-k]}$
$T_e = 334.5 \text{ [K]}$	$T_i = 326.3 \text{ [K]}$	$T_m = 330.4 \text{ [K]}$	$T_{si} = 340.5 \text{ [K]}$

### Unit Settings: [kJ]/[C]/[kPa]/[kg]/[degrees]

$A_a = 2.249 \text{ [m}^2\text{]}$	$A_r = 0.1137 \text{ [m}^2\text{]}$	$C_p = 2061 \text{ [J/kg-K]}$	$d = 0.005 \text{ [m]}$
$L = 5.172 \text{ [m]}$	$\dot{m} = 0.0882 \text{ [kg/s]}$	$Re = 831.9$	$\rho = 900 \text{ [kg/m}^3\text{]}$
$\Delta p = 928982 \text{ [Pa]}$	$D_c = 0.025 \text{ [m]}$	$f_{coil} = 8.012E-02$	$k_c = 400 \text{ [W/m-K]}$
$t = 0.001 \text{ [m]}$	$v = 4.991 \text{ [m/s]}$		